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## BIFURCATED OUTLET GUIDE VANES

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3 [0001] The U.S. Government may have certain rights in this invention pursuant to contract  
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## 6 BACKGROUND OF THE INVENTION

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8 [0002] The present invention relates generally to gas turbine engines, and, more specifically,  
9 to outlet guide vanes therein.

10 [0003] In a gas turbine engine, air is pressurized in a compressor and mixed with fuel for  
11 generating hot combustion gases from which energy is extracted in turbine stages. A high  
12 pressure turbine (HPT) immediately follows the combustor and extracts energy for powering  
13 the compressor. A low pressure turbine (LPT) follows the HPT and extracts additional energy  
14 from the combustion gases for powering an upstream fan in an exemplary aircraft turbofan  
15 engine application.

16 [0004] Each turbine stage includes a row of nozzle vanes specifically configured for  
17 precisely directing the combustion gases into a cooperating row of turbine rotor blades  
18 disposed downstream therefrom. The vanes and blades have specifically configured  
19 aerodynamic profiles for maximizing energy extraction from the combustion gases, with the  
20 profiles thereof being opposite to each other and alternating from stage to stage.

21 [0005] From the last turbine stage in the LPT, the combustion gases are exhausted through  
22 outlet guide vanes (OGVs) typically found in the turbine rear frame immediately downstream  
23 of the LPT.

24 [0006] The OGVs typically have specific aerodynamic profiles to remove swirl, or deswirl  
25 the exhaust flow prior to discharge from the engine for enhancing the performance thereof.  
26 Exhaust swirl is defined as the angle of discharge from the last stage turbine blades relative to  
27 the axial centerline axis of the engine. The swirl angle will vary during low to high power  
28 operation of the engine.

1 [0007] The range or swing in swirl angle varies from minimum to maximum values  
2 depending upon the configuration and operation of the specific engine and may be relatively  
3 small or relatively large. For small values of swirl range, the individual OGVs may have  
4 suitable aerodynamic profiles with generally convex suction sides and generally concave  
5 pressure sides, with a corresponding pitch or angular orientation around the radial axis for  
6 deswirling the exhaust flow. Deswirling operation of the OGVs remains effective as long as  
7 the exhaust flow remains attached to the surfaces of the vanes.

8 [0008] In applications containing large swirl range, the specific aerodynamic profile and  
9 angular orientation of the OGVs may be insufficient to prevent flow separation from the vanes  
10 at one or both extremes in the range of swirl angles. Since a vane is typically optimized for a  
11 specific design point, off-design point operation of the vane changes the aerodynamic  
12 performance thereof eventually leading to flow separation at excess swirl angles of the  
13 exhaust.

14 [0009] Flow separation of the exhaust flow from the OGVs is undesirable since it destroys  
15 the ability of the vanes to properly deswirl the exhaust flow, and therefore reduces  
16 aerodynamic performance and efficiency of the engine.

17 [0010] The ability to deswirl exhaust flow is made more difficult in variable cycle gas  
18 turbine engines such as those specifically configured for short takeoff and vertical landing  
19 (STOVL) operations. STOVL aircraft are typically used by the military for the extreme  
20 military requirements thereof. One type of STOVL aircraft includes an augmented turbofan  
21 engine having an afterburner at the aft end thereof, with a variable area exhaust nozzle. The  
22 afterburner permits additional fuel to be burned therein for substantially increasing the  
23 available thrust and power generated by the engine when required.

24 [0011] Since the afterburner is disposed downstream from the turbine OGVs, performance  
25 of those vanes is further important to ensure suitably deswirled exhaust flow to the afterburner  
26 for the proper performance thereof during reheat or wet operation.

27 [0012] Performance of the turbine OGVs is further complicated by the modification of the  
28 turbofan engine for the STOVL operation which may include an extension of the fan drive  
29 shaft for powering an auxiliary fan mounted in the aircraft wing for enhancing vertical lift.  
30 And, bleed tubes may join the turbofan bypass duct for bleeding therefrom when desired a

1 portion of the fan air which is diverted to corresponding nozzles in the aircraft for providing  
2 additional vertical lift capability and stability control of the aircraft in the STOVL mode of  
3 operation.

4 [0013] Accordingly, this exemplary form of STOVL turbofan engine creates a large swing  
5 or range in the swirl angle of the exhaust discharged from the core engine through the OGVs.  
6 In conventional takeoff and landing operation of the engine, the swirl angle of the exhaust  
7 flow is limited in value and range. Whereas, during the STOVL mode of operation of the  
8 engine, the swirl angle of the exhaust flow from the core engine is substantially changed to  
9 large values.

10 [0014] The typical fixed-design deswirling outlet guide vane is thusly severely limited in its  
11 ability to handle the large range of swirl angle change found in a STOVL aircraft engine.

12 [0015] It is therefore desired to provide outlet guide vanes specifically configured for  
13 accommodating large swing in swirl without undesirable flow separation therein.

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#### 15 BRIEF DESCRIPTION OF THE INVENTION

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17 [0016] A turbine rear frame includes a row of outlet guide vanes extending between outer  
18 and inner bands. Each of the vanes is bifurcated into a forward prow integrally joined to an aft  
19 stem by a septum therebetween. The prow and stem collectively define the aerodynamic  
20 profile of each vane which is locally interrupted at the septum.

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#### 22 BRIEF DESCRIPTION OF THE DRAWINGS

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24 [0017] The invention, in accordance with preferred and exemplary embodiments, together  
25 with further objects and advantages thereof, is more particularly described in the following  
26 detailed description taken in conjunction with the accompanying drawings in which:

27 [0018] Figure 1 is an axial schematic view of an exemplary STOVL turbofan aircraft engine  
28 including a row of outlet guide vanes at the discharge end of the core engine thereof.

29 [0019] Figure 2 is a partly sectional, isometric view of a portion of the OGVs illustrated in  
30 Figure 1 in accordance with an exemplary embodiment.

1 [0020] Figure 3 is a planiform sectional view through some of the OGVs illustrated in  
2 Figures 1 and 2 located immediately downstream of the last stage turbine blades.

3 [0021] Figure 4 is a planiform sectional view, like Figure 3, of the OGVs in accordance with  
4 another embodiment.

5 [0022] Figure 5 is a planiform sectional view, like Figure 3, of the OGVs in accordance with  
6 another embodiment.

7 [0023] Figure 6 is a planiform sectional view, like Figure 3, of the OGVs in accordance with  
8 another embodiment.

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10 DETAILED DESCRIPTION OF THE INVENTION

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12 [0024] Illustrated schematically in Figure 1 is a turbofan gas turbine engine 10 specifically  
13 configured for powering a STOVL aircraft in an exemplary application. The engine is  
14 axisymmetrical about a longitudinal or axial centerline axis and includes in serial flow  
15 communication a fan 12, multistage axial compressor 14, combustor 16, high pressure turbine  
16 (HPT) 18, and low pressure turbine (LPT) 20. The HPT 18 is joined to the compressor 14 by  
17 one shaft, and the LPT 20 is joined to the fan 12 by another shaft.

18 [0025] During operation, air 22 enters the engine and is pressurized in the compressor 14  
19 and mixed with fuel in the combustor 16. The aspirated air is ignited for generating hot  
20 combustion gases 24 which are discharged in turn through the HPT 18 and LPT 20 that extract  
21 energy therefrom. The HPT powers the compressor, and the LPT powers the fan.

22 [0026] In the exemplary STOVL configuration illustrated in Figure 1, the engine also  
23 includes an augmentor or afterburner 26 in which additional fuel may be burned when desired  
24 for increasing the exhaust thrust from the engine. A variable area nozzle 28 is located at the  
25 aft end of the afterburner and cooperates therewith for maximizing performance of the engine  
26 over its intended flight envelope.

27 [0027] For the STOVL application, the engine 10 illustrated in Figure 1 further includes an  
28 auxiliary fan 30 operatively joined to the main fan 12 by a drive shaft extension thereof. And,  
29 large bleed tubes 32 are joined in flow communication to the upstream end of the bypass duct  
30 surrounding the core engine for bleeding a portion of the fan air therefrom when desired.

1 [0028] The STOVL engine 10 illustrated schematically in Figure 1 may have any  
2 conventional configuration and operation for operating in a conventional mode without the  
3 use of the auxiliary fan 32 and bleed tubes 32, and in a STOVL mode of operation in which  
4 the auxiliary fan 30 is powered by the engine, and fan air is bled from the engine through the  
5 tubes 32 to suitable nozzles in the aircraft for providing additional lift and stability control  
6 thereof during operation. However, the STOVL capability of the engine 10 illustrated in  
7 Figure 1 results in a substantial range or swing in swirl angles of the exhaust flow 24  
8 discharged from the LPT 20 during operation into the augmentor 26.

9 [0029] Accordingly, the engine includes a turbine rear frame 34 specifically configured for  
10 accommodating the extended range in swirl angle for this type of engine without the need for  
11 mechanical articulation thereof which would otherwise increase complexity and weight of the  
12 engine. The rear frame 34 is illustrated isometrically in part in Figure 2 and in planiform view  
13 in part in Figure 3 in accordance with an exemplary embodiment.

14 [0030] The rear frame is an annular assembly of components and is axisymmetrical about  
15 the longitudinal or axial centerline axis of the engine. The frame includes a plurality of outlet  
16 guide vanes (OGVs) 36 arranged in a circumferential row extending radially between outer  
17 and inner supporting bands 38,40.

18 [0031] As shown in Figure 3, the vanes 36 are disposed immediately downstream of the last  
19 stage row of turbine rotor blades 42 found in the LPT, and which extend radially outwardly  
20 from their supporting rotor disk. During operation, the exhaust flow 24 is discharged from the  
21 turbine blades 42 with a suitable swirl angle A which is measured relative to the axial  
22 centerline axis of the engine, toward the OGVs.

23 [0032] The swirl angle of the exhaust flow varies from a maximum positive value  
24 represented by the angle A to a corresponding minimum value represented by the negative  
25 swirl angle -B. For example, the maximum swirl angle may be about +40 degrees, and the  
26 minimum swirl angle may be about -20 degrees, with a combined range or swing of swirl  
27 being the 60 degree combination thereof.

28 [0033] The large range in swirl angle is specifically due to the variable cycle configuration  
29 of the STOVL engine 10 illustrated in Figure 1. During normal operation of that engine, the  
30 turbofan engine operates in the normal manner of a turbofan engine with the core exhaust flow

1 and the bypass air being discharged through the augmentor to the common outlet nozzle.  
2 And, during STOVL operation of the engine, the fan bypass flow around the core engine is  
3 temporarily interrupted by a suitable valve to divert fan air through the bleed tubes 32, while  
4 the auxiliary fan 30 is engaged for diverting corresponding power from the engine.

5 [0034] In this STOVL operation of the engine, the swirl angle of the exhaust flow 24 is  
6 drastically altered from its normal range as the exhaust flow is nevertheless discharged  
7 between the OGVs 36 into the augmentor.

8 [0035] In order to accommodate the large range in swirl angle of about 60 degrees, and even  
9 larger, the OGVs 36 illustrated in Figure 3 are bifurcated in a specific manner for  
10 accommodating the large change in direction of the incident exhaust flow.

11 [0036] More specifically, each of vanes 36 illustrated in Figure 3 is bifurcated into a forward  
12 prow or nose segment 44 and an aft stern or tail segment 46 by a narrow septum 48 extending  
13 chordally or axially therebetween. The prow and stern are an integral, and preferably unitary  
14 assembly and collectively define the aerodynamic profile or perimeter of each vane 36 with a  
15 convex suction side 50 and a circumferentially opposite, concave pressure side 52. The two  
16 sides 50,52 extend chordally between a leading edge 54 at the front of the prow 44 and an  
17 axially opposite trailing edge 56 at the aft end of the stern 46.

18 [0037] As shown in Figure 3, the combined configuration of the prow and stern includes a  
19 chord extending between the leading and trailing edges thereof which may be located at a  
20 suitable pitch angle C relative to the axial centerline axis of the engine. The aerodynamic  
21 contour of the vanes 36 is generally opposite to the contours of the last stage turbine blades 42,  
22 and have a suitable pitch angle C for maximizing the deswirling capability of the vanes  
23 corresponding with the maximum expected swirl angle from the blades in the preferred  
24 embodiment.

25 [0038] Since the prow and stern illustrated in Figure 3 are separated from each other by the  
26 joining septum or ligament 48, these two portions of each vane may be separately tailored in  
27 profile while still collectively providing the overall aerodynamic profile of the vane. The  
28 overall profile of the vane is locally interrupted chordally between the prow and stern at the  
29 septum therebetween. The septum 48 is relatively narrow across the width of the vane  
30 between its opposite sides, and introduces a first radial slot 58 which separates in part the

1   prow from the stern. The septum 48 itself is preferably imperforate.

2   **[0039]**   The stern 46 illustrated in Figure 3 defines the major portion of each vane having the  
3   greatest amount of chordal length, whereas the prow 44 is relatively short in chordal length for  
4   the remaining minor portion of each vane. The prow 44 defines the leading edge portion of  
5   each vane, and the stern 46 converges aft to the trailing edge 56 from its junction with the  
6   forward prow at the septum 48. In this way, the prow shields the forward end of the stern and  
7   cooperates therewith as further described hereinbelow for substantially increasing the range of  
8   permissible swirl angle without undesirable flow separation of the exhaust flow over the vanes  
9   during operation.

10   **[0040]**   The vanes illustrated in Figure 3 are hollow at least in part and include in this  
11   exemplary embodiment an internal prow channel 60 extending radially through the prow, and  
12   a corresponding internal stern channel 62 extending radially through the stern. The two  
13   internal channels 60,62 in each vane preferably extend through the outer band 38 as illustrated  
14   in Figure 2 for providing flow communication therethrough.

15   **[0041]**   The first radial slot 58 illustrated in Figure 3 is disposed between the prow and stern  
16   on the suction side of the vane. A second radial slot 64 is disposed in the prow 44 itself on the  
17   opposite pressure side of the vane forward of the septum 48. In this way, the prow and stern  
18   may be specifically configured for introducing the two radial slots 58,64 on opposite sides  
19   thereof immediately downstream of the leading edge.

20   **[0042]**   A row of prow apertures 66 is disposed through the pressure sidewall thereof in flow  
21   communication with the prow channel 60 on one side and the second radial slot 64 on the  
22   opposite side, which slot is fed by the prow channel 60.

23   **[0043]**   Correspondingly, a row of stern apertures 68 is disposed in the front wall of the stern  
24   adjacent to the imperforate septum 48 for providing flow communication between the stern  
25   channel 62 and the first radial slot 58, which slot feeds the stern channel. In this way, the two  
26   slots 58,64 cooperate with the respective internal channels in the stern and prow in flow  
27   communication through the outer band 38 illustrated in Figure 2.

28   **[0044]**   As shown in Figures 1 and 2, suitable means 70 are provided for supplying  
29   pressurized air 22 into the row of hollow vanes 36 for discharge through the prow apertures 66  
30   into the corresponding prow slots 64. For example, the pressurized air may be bled from the

1 compressor 14 or fan 12 by suitable conduits having flow control valves therein to distribute  
2 the pressurized air through an annular manifold surrounding the outer band 38 into each of the  
3 vanes 36 suitably joined thereto.

4 [0045] In this way, a common supply manifold is joined in flow communication to each of  
5 the prow channels 60 through the outer band for providing pressurized air into the prow. The  
6 pressurized air may then be discharged through the second slots 64, on the pressure side of the  
7 vanes for example, to promote and maintain attachment of the exhaust flow over the vane.

8 [0046] Correspondingly, Figures 1 and 2 illustrate additional means 72 for withdrawing the  
9 exhaust flow 24 from the exemplary first slots 58 disposed on the suction side of the vanes as  
10 illustrated in Figure 3. As the exhaust flows over the vane suction sides during operation, a  
11 portion thereof is extracted or withdrawn through the stern apertures 68 and into the stern  
12 channel 62 for discharge through the outer band.

13 [0047] As shown in Figures 1 and 2, the withdrawing means 72 may include another annular  
14 manifold surrounding the outer band and disposed in flow communication with the  
15 corresponding stern channels 62 in each of the vanes for extracting exhaust flow therefrom.  
16 The withdrawing manifold may be simply vented to the atmosphere externally of the engine.  
17 During aircraft flight, the pressure outside the engine is substantially lower than the pressure  
18 inside the engine of the exhaust flow between the outlet guide vanes, and the atmosphere  
19 provides a suitable sink for withdrawing exhaust flow from the vanes.

20 [0048] The exemplary first slots 58 therefore draw the exhaust flow over the suction side of  
21 the prow for maintaining flow attachment thereto and preventing undesirable flow separation  
22 of the exhaust as it flows downstream over the suction side of the stern during operation.

23 [0049] Figure 3 illustrates one embodiment of the bifurcated vanes 36 with specifically  
24 configured prow 44 and stern 46. The introduction of the suction side radial slot 58 interrupts  
25 the axial continuity of the vane suction side, and correspondingly introduces a locally small  
26 convex suction side on the prow itself leading into the slot 58. The convex profile of the prow  
27 itself may be used for enhancing flow attachment of the exhaust flow thereover as well as over  
28 the remaining suction side of the vane over the stern.

29 [0050] In this way, the bifurcated vane may be designed to handle the large range of swirl  
30 angles found in the STOVL aircraft engine without requiring articulation or repositioning of

1 the vane itself, and the associated complexity thereof. The stern channel 62 may be simply  
2 vented outside the engine so that a portion of the exhaust flow over the vane suction side is  
3 withdrawn through the first slot 58 for enhancing flow attachment notwithstanding large  
4 variation in the swirl angle.

5 [0051] Correspondingly, the second slot 64 receives pressurized air from the prow channel  
6 60 and discharges that air in a thin film aft along the pressure side of the vane for enhancing  
7 flow attachment of the exhaust flow thereover.

8 [0052] The different configurations of the prow 44 and stern 46 and the associated slots  
9 58,64 permit various permutations thereof which may be used to advantage in increasing the  
10 range of swirl angle while reducing or avoiding undesirable flow separation over the vanes  
11 during operation. Figure 3 illustrates one embodiment, and Figures 4, 5, and 6 illustrate  
12 alternate embodiments in which common features are identified by common reference  
13 numerals, and suitably modified for the different embodiments. Since the prows and sterns  
14 have modified configurations in Figures 4-6, they themselves are differently numbered,  
15 notwithstanding the otherwise similar features and operation thereof.

16 [0053] For example, the first, or suction-side, slot 58 illustrated in Figure 3 faces forward  
17 toward the leading edge in the vane suction side 50 for collecting a portion of the exhaust flow  
18 therein for discharge through the outer band. In Figure 4, the modified prow 74 is similarly  
19 configured for introducing the forward facing first slot 58. And, in Figure 5 the modified  
20 prow 78 is also similarly configured for introducing the forward facing first slot 58.

21 [0054] In the alternate embodiments illustrated in Figures 3, 4, and 5, the prows and sterns  
22 therein are spaced chordally apart at the corresponding first slots 58 in the suction sides to  
23 provide unobstructed open access to those slots for freely receiving the exhaust flow.  
24 Correspondingly, the suction side portions of those differently configured prows 44,74,78  
25 have locally convex profiles for maintaining flow attachment of the incoming exhaust flow  
26 irrespective of the large range in swirl angles.

27 [0055] Figure 6 illustrates that the first slot 58 in the suction side of the vane may  
28 alternatively face aft toward the trailing edge, with the corresponding prow 82 including a lip  
29 86 extending aft over most of the first slot 58.

30 [0056] Figure 5 illustrates that the second slot 64 in the vane pressure side 52 may

1 alternatively face forward toward the leading edge 54 for collecting the exhaust if desired.  
2 And, the prow 78 and stern 80 are spaced chordally apart at the forward facing slot 64 to  
3 provide unobstructed open access thereto.

4 [0057] Figures 4 and 6 illustrate additional modifications in which the second slot 64 faces  
5 aft toward the trailing edge in the vane pressure side 52. And, additional lips 86 extend aft  
6 over the corresponding second slots 64 to provide smooth transitions between the  
7 corresponding prows and sterns.

8 [0058] Figures 4-6 illustrate exemplary embodiments in which the corresponding septums  
9 48 thereof are spaced inwardly from both sides of the vanes to introduce opposite radial slots  
10 58,64 therein.

11 [0059] In the Figure 5 embodiment, the two slots 58,64 are similarly configured in the  
12 corresponding suction and pressure sides 50,52 of the vane, and both face forward toward the  
13 leading edge without flow obstruction. In this embodiment, the prow 78 is solid without any  
14 internal flow channel, and the stern 80 alone includes the stern channel 62 and two rows of the  
15 stern apertures 68 corresponding with the two radial slots 58,64 in the opposite sides of the  
16 vane. In this way, corresponding portions of the exhaust flow 24 may be withdrawn from  
17 both sides of the vane just aft of the leading edge for promoting flow attachment on both sides  
18 of the vane notwithstanding the large swing in swirl angle.

19 [0060] Figure 6 illustrates yet another embodiment in which the two slots 58,64 in the  
20 opposite suction and pressure sides 50,52 of the vane both face aft toward the trailing edge,  
21 with each slot including a corresponding lip 86 for maintaining an aerodynamically smooth  
22 junction between the prow 82 and stern 84.

23 [0061] In this embodiment, the stem 84 immediately aft of the septum 48 may be solid  
24 without the internal stern channel therein, and the prow 82 includes a common prow channel  
25 60 for feeding both radial slots 58,64 through corresponding rows of the prow apertures 66.  
26 The prow 82 including the aft lips 86 thereof may be specifically configured in profile for  
27 enhancing flow attachment of the exhaust flow during operation, with flow attachment being  
28 further enhanced by the discharge of pressurized air from the two slots 58,64 during operation.

29 [0062] Figure 4 illustrates yet another embodiment in which the two slots 58,64 are disposed  
30 on opposite sides of the common septum 48. The first slot 58 of the vane suction side 50 faces

1 forward toward the leading edge without obstruction, and the second slot 64 in the vane  
2 pressure side 52 faces aft toward the trailing edge, and covered in most part by the lip 86. The  
3 lip 86 in this embodiment overlaps the pressure side of the stern 76, and is not aligned flush  
4 therewith in the manner illustrated in the Figure 6 embodiment.

5 [0063] In the Figure 3 embodiment, the septum 48 is spaced inwardly from only the vane  
6 suction side 50 and adjoins the vane pressure side 52. And, the first radial slot 58 in the vane  
7 suction side faces forward, whereas the second slot 64 in the prow pressure side 52 faces aft  
8 toward the trailing edge.

9 [0064] In the Figure 3 embodiment, the prow 44 itself includes the second slot 64 facing aft  
10 in the pressure side of the vane upstream from the septum 48. And, in the alternate  
11 embodiments illustrated in Figures 4 and 6, the aft-facing second slot 64 is located at the  
12 septum 48 itself under the aft lip extension 86 of the prow.

13 [0065] But for the two rows of prow and stern apertures 66,68 which provide flow  
14 communication between the respective slots 58,64 and prow and stern channels 60,62, the  
15 various embodiments of prows and sterns are otherwise imperforate for maintaining  
16 aerodynamically smooth contours of the bifurcated vanes.

17 [0066] In the several embodiments illustrated in Figures 3-6, the prows and sterns may be  
18 formed together in a common casting in view of the complex configuration thereof. The  
19 internal channels, apertures, and slots may also be conveniently formed by conventional  
20 casting.

21 [0067] The aft portions of the various sterns illustrated in Figures 3-6 are preferably  
22 manufactured as separately fabricated sheet metal components and suitably joined to the  
23 corresponding castings by brazing for example. Alternatively, the entire bifurcated vane may  
24 be cast in one unitary component, or could alternatively be a fabrication of sheet metal parts  
25 integrally joined together in a one-piece component or assembly.

26 [0068] The various embodiments of the bifurcated outlet guide vanes illustrated in the  
27 several figures introduce corresponding prows and sterns separated by radial slots in the  
28 corresponding sides of the vanes. The prow may be separately configured for maximizing  
29 aerodynamic performance thereof based on the particular incident angle of exhaust flow, with  
30 the corresponding sterns being separately configured for maintaining flow attachment of the

1 exhaust flow during the deswirling process.

2 [0069] The slots in the pressure and suction sides of the vanes may be configured for  
3 discharging pressurized air along the corresponding vane side or withdrawing a portion of the  
4 exhaust flow for maintaining flow attachment without undesirable flow separation during  
5 operation. Pressurized air may be introduced on one or both sides of each vane; or the exhaust  
6 flow may be withdrawn from one or both sides of each vane; or air may be supplied on one  
7 side while exhaust flow is withdrawn on the other side of each vane as desired for maximizing  
8 performance.

9 [0070] The corresponding means for supplying pressurized air or withdrawing exhaust flow  
10 from the outlet guide vanes may be suitably coordinated using flow control valves under  
11 computer control for best coordinating operation of the vanes with operation of the engine  
12 from the normal mode of operation to the STOVL mode of operation during which the swirl  
13 angle of the exhaust flow discharged from the core engine swings over a substantially large  
14 range, such as the 60 degrees range disclosed above, or even higher.

15 [0071] While there have been described herein what are considered to be preferred and  
16 exemplary embodiments of the present invention, other modifications of the invention shall be  
17 apparent to those skilled in the art from the teachings herein, and it is, therefore, desired to be  
18 secured in the appended claims all such modifications as fall within the true spirit and scope of  
19 the invention.

20 [0072] Accordingly, what is desired to be secured by Letters Patent of the United States is  
21 the invention as defined and differentiated in the following claims in which I claim: